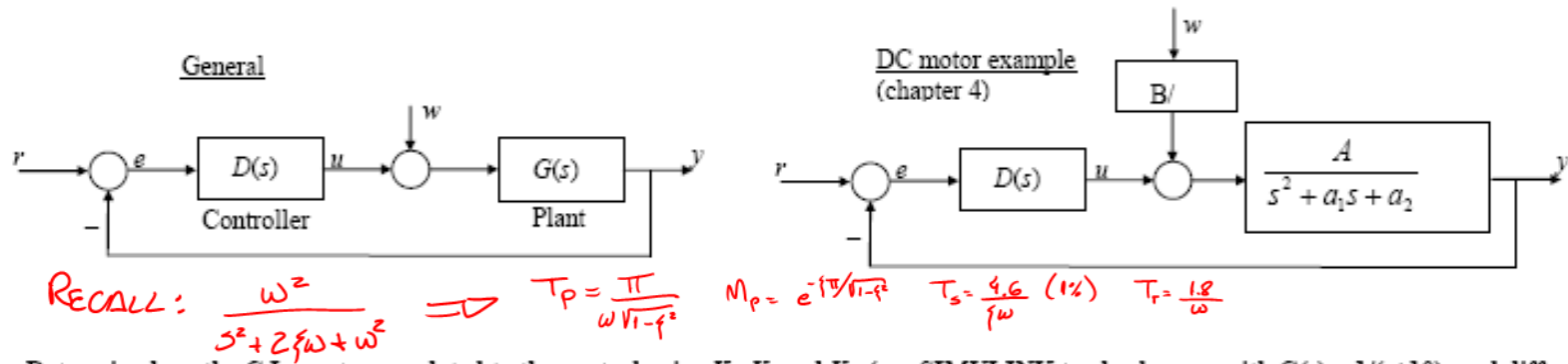


# LECTURE 21

## 4.5 PID CONTROL



**Objective:** Determine how the C.L. roots are related to the control gains  $K_p$ ,  $K_I$  and  $K_d$  (use SIMULINK to check some with  $G(s) = 1/(s+10)$ , and different  $D(s)$ )

Control Type	Time Domain Representation	Frequency domain representation of controller	Characteristic equation (denom. of closed loop TF=0), $1 + D(s)G(s) = 0$	Remarks
P	$u(t) = K_p e(t)$	$D(s) = K_p$	$s^2 + a_1s + a_2 + AK_p = 0$	*1 degree of freedom, and so cannot locate the roots at any location. *Can reduce error magnitude but non-zero steady-state error exists. *Can increase the speed of response but has much larger transient overshoot.
I	$u(t) = K_I \int_{t_0}^t e(t) dt$	$D(s) = \frac{K_I}{s}$	$s^3 + a_1s^2 + a_2s + AK_I = 0$	*Can reduce or eliminate constant steady-state errors at the cost of worse transient response.
PI	$u(t) = \left( K_p e(t) + K_I \int_{t_0}^t e(t) dt \right)$	$D(s) = \frac{K_p s + K_I}{s}$	$s^3 + a_1s^2 + (a_2 + AK_p)s + AK_I = 0$	*2 degrees of freedom and the designer can choose $K_p$ and $K_I$ independently to provide better transient response. *Approximated by lag controller.
D	$u(t) = K_d \dot{e}(t)$	$D(s) = K_d s$	$s^2 + (a_1 + AK_d)s + a_2 = 0$	*Pure derivative feedback is not practical to implement.
PD	$u(t) = (K_p e(t) + K_d \dot{e}(t))$	$D(s) = (K_p + K_d s)$	$s^2 + (a_1 + AK_d)s + (a_2 + AK_p) = 0$	*Can increase the damping and generally improve the stability of a system. *Approximated by lead controller.
PID	$u(t) = \left( K_p e(t) + K_I \int_{t_0}^t e(t) dt + K_d \dot{e}(t) \right)$	$D(s) = \left( K_p + \frac{K_I}{s} + K_d s \right)$	$s^3 + (a_1 + AK_d)s^2 + (a_2 + AK_p)s + AK_I = 0$	*3 degrees of freedom and can provide an acceptable degree of error reduction simultaneously with acceptable stability and damping

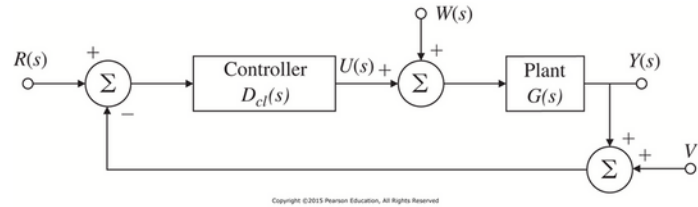
Other controller types: state feedback, adaptive, robust, stochastic, neural, fuzzy,...

RECALL :

$$Y = TR + GSW - TV$$

$$E = SR - GSW + TV$$

$$(E = R - Y)$$



$$T = \frac{GD}{1+GD}$$

$$S = \frac{1}{1+GD}$$

$$T = 1 - S$$

OPEN LOOP

$$G = \frac{b}{a}$$

$$D = \frac{c}{d}$$

$$T = GD = \frac{bc}{ad}$$

IF  $G$  IS UNSTABLE - I.E. ROOTS (POLES) OF  $\underline{a}$  IN RHP - ONE MIGHT THINK THAT THE SAME ROOTS IN  $\underline{c}$  (ZEROS) COULD CANCEL THOSE POLES & UNSTABLE (SENSITIVITY!) \*

SIMILARLY, IF  $G$  HAS POOR RESPONSE DUE TO SOME ZEROS OF  $\underline{b}$  IN RHP, POLES OF  $\underline{d}$  TO CANCEL THOSE ZEROS WOULD, AGAIN, LEAD TO AN UNSTABLE SYSTEM.

\* SENSITIVITY

$$S_G^{T_{ol}} \triangleq \frac{\frac{\delta T_{ol}}{T_{ol}}}{\frac{\delta G}{G}} = \frac{\cancel{\delta G}}{\frac{\delta G}{G}} = 1$$

10% in  $G \Rightarrow 10\%$  in  $T_{ol}$

## FEEDBACK

$$G = \frac{b}{a} \quad D = \frac{c}{d} \quad T = \frac{\frac{bc}{ad}}{1 + \frac{bc}{ad}} = \frac{bc}{ad + bc}$$

Now, IF  $G$  IS UNSTABLE - POLES OF  $a$  IN RHP -  
THE EXACT ~~SOME~~ ZEROS IN  $b$  WILL NOT CANCEL  
THE POLES  $\Rightarrow$  HOW TO CANCEL UNSTABLE POLES?  
BY ADDING STABLE ZEROS ?!

$$\text{Ex: } G = \frac{b}{a} = \frac{1}{s^2 - 1} = \frac{1}{(s+1)(s-1)}$$

$$D = \frac{c}{d} = \frac{k(s+1)}{(s+\delta)}$$

$$\Rightarrow T = \frac{bc}{ad + bc} = \frac{\cancel{k(s+1)}}{(s+1)(s-1)(s+\delta) + \cancel{k(s+1)}}$$

$$T = \frac{k}{(s-1)(s+\delta) + k} = \frac{k}{s^2 + (\delta-1)s + (k-\delta)}$$

ROUTH CRITERIA  $\Rightarrow$  CHOICE OF  $k, \delta$   
( $k > \delta > 1$ )

Also,

$$T_{CL} + \delta T_{CL} = \frac{(G + \delta G)D}{1 + (G + \delta G)D}$$

$$\delta T_{CL} \approx \frac{dT_{CL}}{dG} \delta G$$

$$S_G^{T_{CL}} \triangleq \frac{\frac{\delta T_{CL}}{T_{CL}}}{\frac{\delta G}{G}} = \frac{G}{T_{CL}} \frac{\delta T_{CL}}{\delta G} \approx \frac{G}{T_{CL}} \frac{dT_{CL}}{dG} =$$

$$S_G^{T_{CL}} = \frac{G}{T_{CL}} \frac{d}{dG} \left( \frac{GD}{1+GD} \right) = \dots = \frac{1}{1+GD}$$

if GAIN OF  $1+GD = 100 \Rightarrow 10\%$  in  $G \Rightarrow 0.1\%$  in  $T_{CL}$

$$\text{NOTE: } S_G^{T_{CL}} = S = \frac{1}{1+GD}$$

# System Type (FOR TRACKING THE INPUT)

K | For  $R(t) = t^k \therefore e_{ss} = \lim_{t \rightarrow \infty} e(t) = \underline{cte}$  (FOR  $w = v = 0$ )

$$e_{ss} = \lim_{s \rightarrow 0} s e(s) = \underline{cte} \quad \text{if roots } e(t) \in \text{LHP}$$

$$e_{ss} = \lim_{s \rightarrow 0} s R = \lim_{s \rightarrow 0} \frac{s}{1+GD} * \frac{1}{s^{k+1}} = \lim_{s \rightarrow 0} \frac{1}{1+GD} * \frac{1}{s^k}$$

if  $D = \frac{c}{d} = \frac{c}{s^n}$   $\nearrow$  n integrators

$$e_{ss} = \lim_{s \rightarrow 0} \frac{1}{1 + \frac{GC}{s^n}} * \frac{1}{s^k} = \frac{s^n}{s^n + GC} * \frac{1}{s^k}$$

IF  $GC(s=0) = K_n \neq 0$  SYSTEM TYPE = n

RECALL :

n = 2

<u>INPUT</u> :	STEP	RAMP	PARABOLE
<u>e<sub>ss</sub></u> :	0	0	cte

# System Type (FOR REGULATION - DISTURBANCE REJECTION)

(FOR  $R=V=0$ )

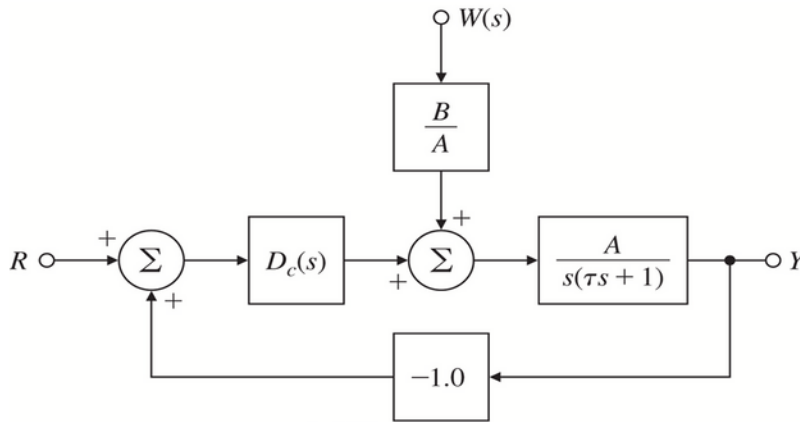
$K$  | For  $W(t) = t^k \therefore e_{ss} = \lim_{t \rightarrow \infty} e(t) = c t^k$

$e(s) = R(s) - Y(s) = -Y(s) \Rightarrow \frac{e(s)}{W(s)} = -\frac{Y(s)}{W(s)} = T_W(s)$   
 $\hookrightarrow = GS$  for unit feedback

taking  $\lim_{t \rightarrow \infty}$  or  $\lim_{s \rightarrow 0}$

$Y_{ss} = \lim_{s \rightarrow 0} s T_W(s) * \frac{1}{s^{k+1}} = \lim_{s \rightarrow 0} T_W(s) \frac{1}{s^k}$

Ex



$D = K_P \quad \& \quad D = K_P + \frac{K_I}{s}$

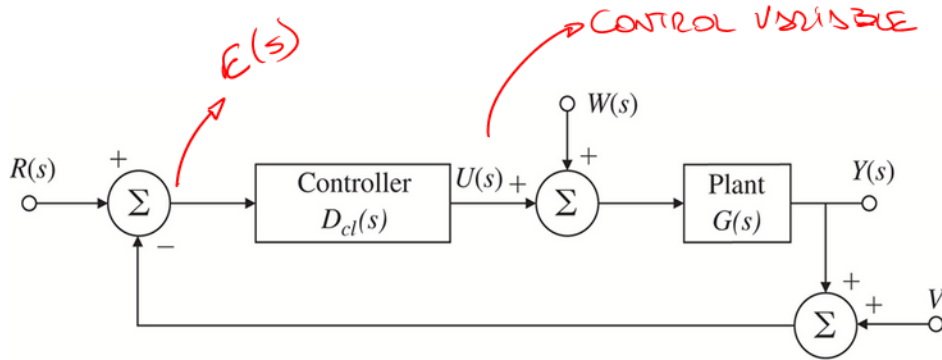
$\Downarrow$   
type 0

$Y_{ss} = \frac{-B}{AK_P}$

$\Downarrow$   
type 1

$Y_{ss} = \frac{-B}{AK_I}$

# PROPORTIONAL CONTROL



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$u(t) = K_p e(t)$  = control is proportional to error

$$\frac{U(s)}{E(s)} = D_{cl}(s) = K_p \quad \neq \quad T(s) = \frac{K_p G(s)}{1 + K_p G(s)} \quad e_{ss} = \lim_{s \rightarrow 0} \frac{s}{1 + K_p G(s)} \approx \frac{1}{s^{k+1}}$$

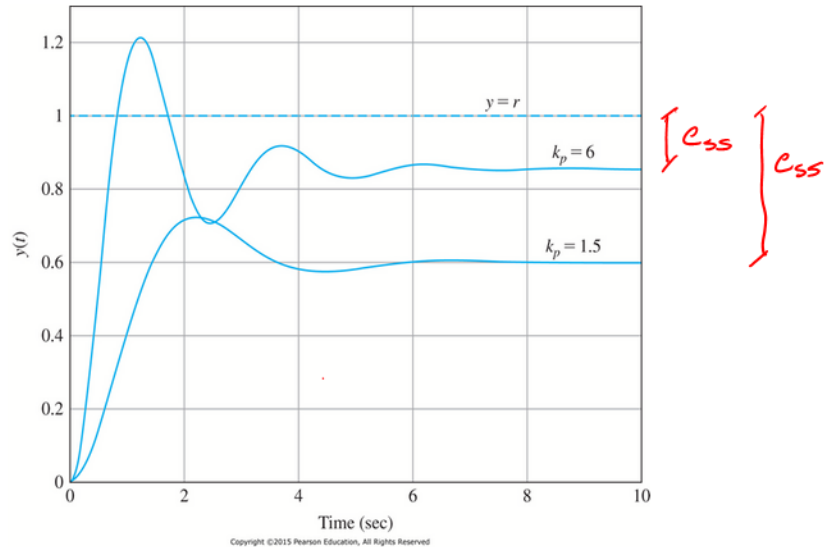
if  $G(s) = \frac{b}{a} = \frac{A}{s^2 + a_1 s + a_2} \quad (D = \frac{c}{d} = K_p)$

$$T(s) = \frac{bc}{ad + bc} = \frac{A}{s^2 + a_1 s + a_2 + K_p A}$$

SYSTEM TYPE FOR TRACKING = 0

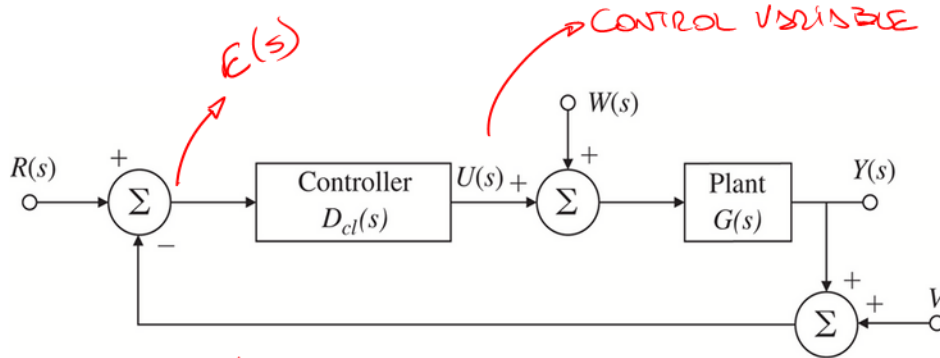
$$\Rightarrow e_{ss} = \frac{1}{a_2 + K_p A}$$

Wn Adjust



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# INTEGRAL CONTROL



$$u(t) = K_I \int_{t_0}^t e(t) dt = \text{CONTROL IS THE ACCUMULATED ERROR}$$

$$\frac{U(s)}{E(s)} = D_{cl}(s) = \frac{K_I}{s} \quad \neq \quad \Delta(s) = \frac{\frac{K_I}{s} G(s)}{1 + \frac{K_I}{s} G(s)} = \frac{K_I G(s)}{s + K_I G(s)}$$

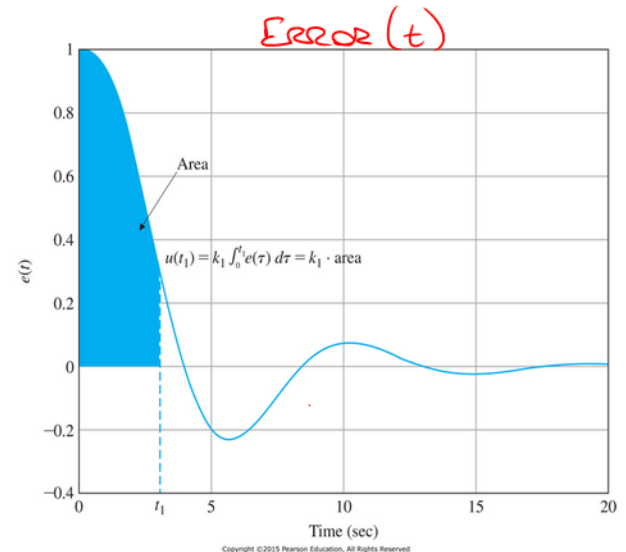
IF  $G(s=0) = 1$

$$e_{ss} = \lim_{s \rightarrow 0} \frac{s}{s + K_I} * \frac{1}{s^k}$$

SYSTEM TYPE FOR TRACKING = 1

$$e_{ss} = 0 \text{ for } R(s) = \frac{1}{s} \text{ (unit step)}$$

$$e_{ss} = \frac{1}{K_I} \text{ for } R(s) = \frac{1}{s^2} \text{ (ramp)}$$



IF  $G(s=0) = 1$  (AS ABOVE)

$$y_{ss} = \lim_{s \rightarrow 0} s T_w(s) * \frac{1}{s^{k+1}} = \lim_{s \rightarrow 0} s \frac{G}{1+DG} * \frac{1}{s^{k+1}} = \lim_{s \rightarrow 0} \frac{Gs}{s+K_I G} + \frac{1}{s^k}$$

$\downarrow \frac{K_I G}{s}$

SYSTEM TYPE FOR REGULATION = 1

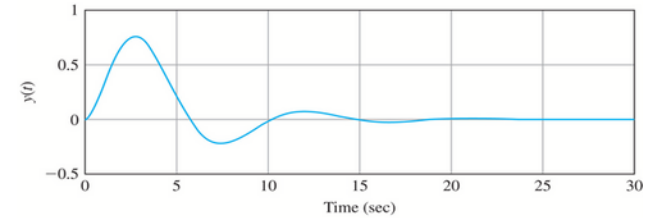
$$y_{ss} = 0 \quad \text{for } w(s) = \frac{1}{s} \text{ (step)}$$

$$y_{ss} = \frac{1}{K_I} \quad \text{for } w(s) = \frac{1}{s^2} \text{ (ramp)}$$

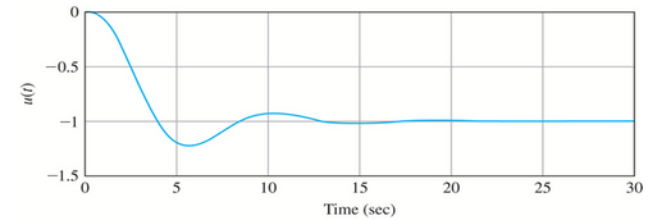
$$u_{ss} = \lim_{s \rightarrow 0} s \frac{-DG}{1+DG} * \frac{1}{s^{k+1}} = \lim_{s \rightarrow 0} -\frac{K_I G}{s+K_I G} * \frac{1}{s^k}$$

$$u_{ss} = -1 \quad \text{for } w(s) = \frac{1}{s} \text{ (step)}$$

$$u_{ss} = \infty \quad \text{for } w(s) = \frac{1}{s^2} \text{ (ramp)}$$



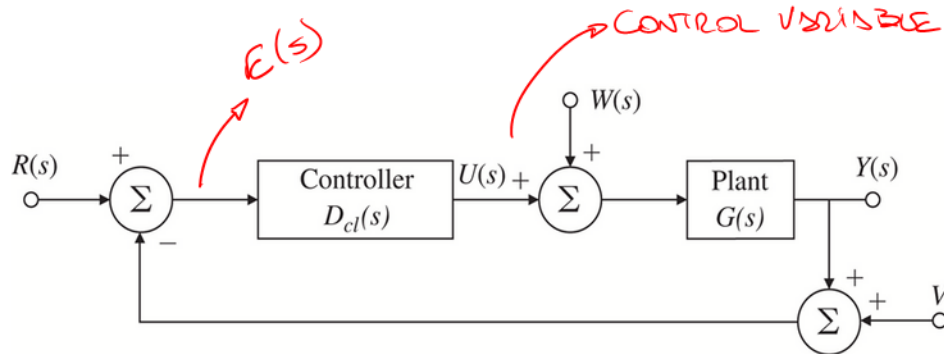
(a)



(b)

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# DERIVATIVE CONTROL

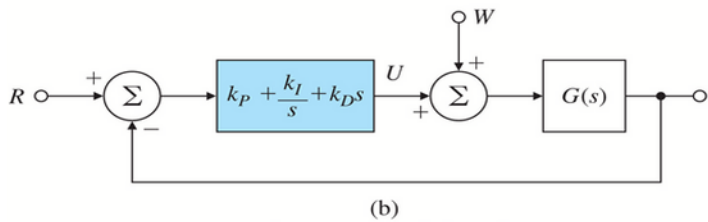
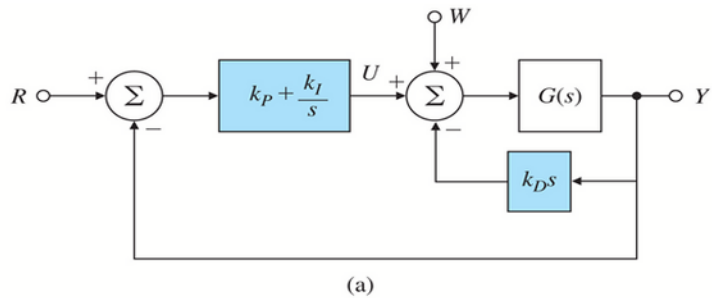


$$u(t) = K_D \dot{e}(t) = \text{CONTROL IS THE RATE OF THE ERROR}$$

$$\frac{U(s)}{E(s)} = D_{cl}(s) = K_D s \quad \# \quad T(s) = \frac{K_D s G(s)}{1 + K_D s G(s)} = \frac{K_D s G(s)}{1 + K_D s G(s)}$$

$$\text{if } G(s) = \frac{b}{a} = \frac{A}{s^2 + a_1 s + a_2}$$

$$T(s) = \frac{bc}{ad + bc} = \frac{AK_D s}{s^2 + (a_1 + K_D A)s + a_2}$$

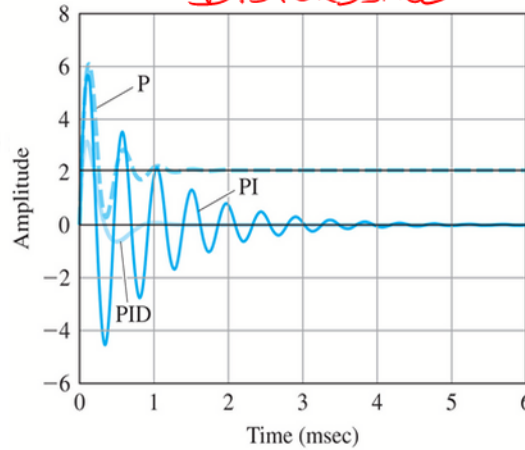


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PID

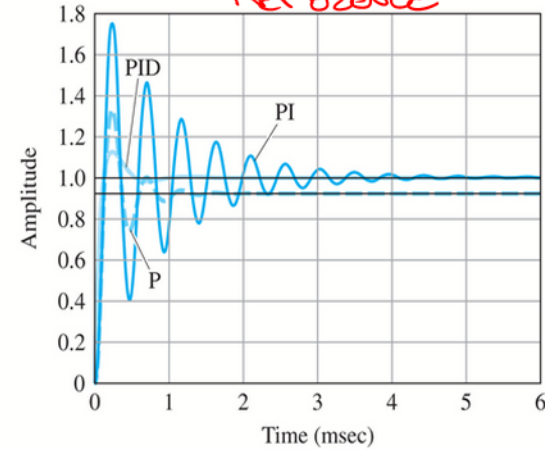
$$G(s) = s^2 + a_1 s + a_2$$

STEP  
DISTURBANCE



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STEP  
REFERENCE



$$u(t) = K_p e(t) + K_I \int_{t_0}^t e(t) dt + K_D \dot{e}(t)$$

$$u(s) = \left( K_p + \frac{K_I}{s} + K_D s \right) e(s)$$